

Development of Peltier Cooled Frost point Hygrometer PCFH

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Stratospheric Water Vapor and the Frost Point Hygrometer

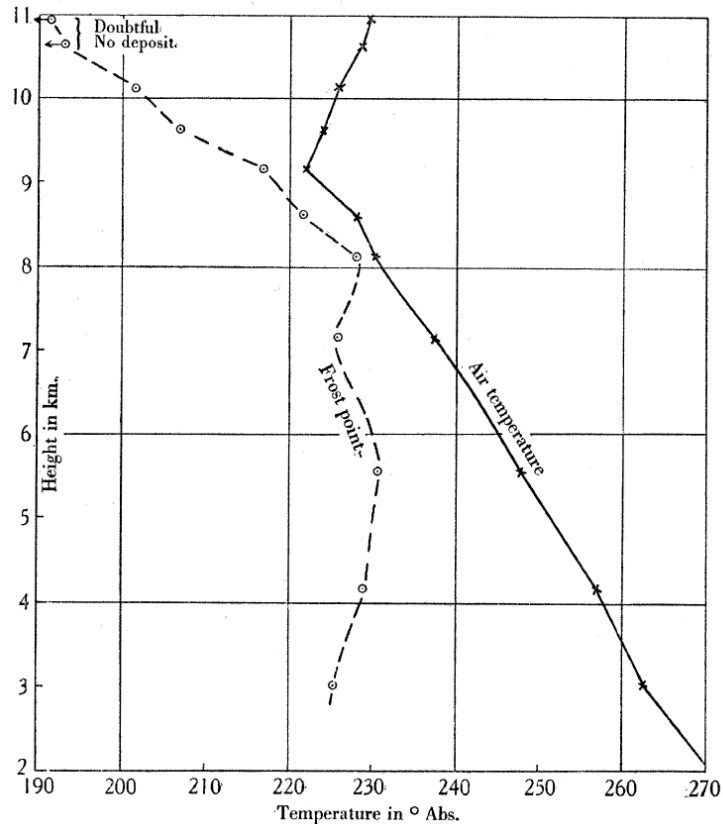
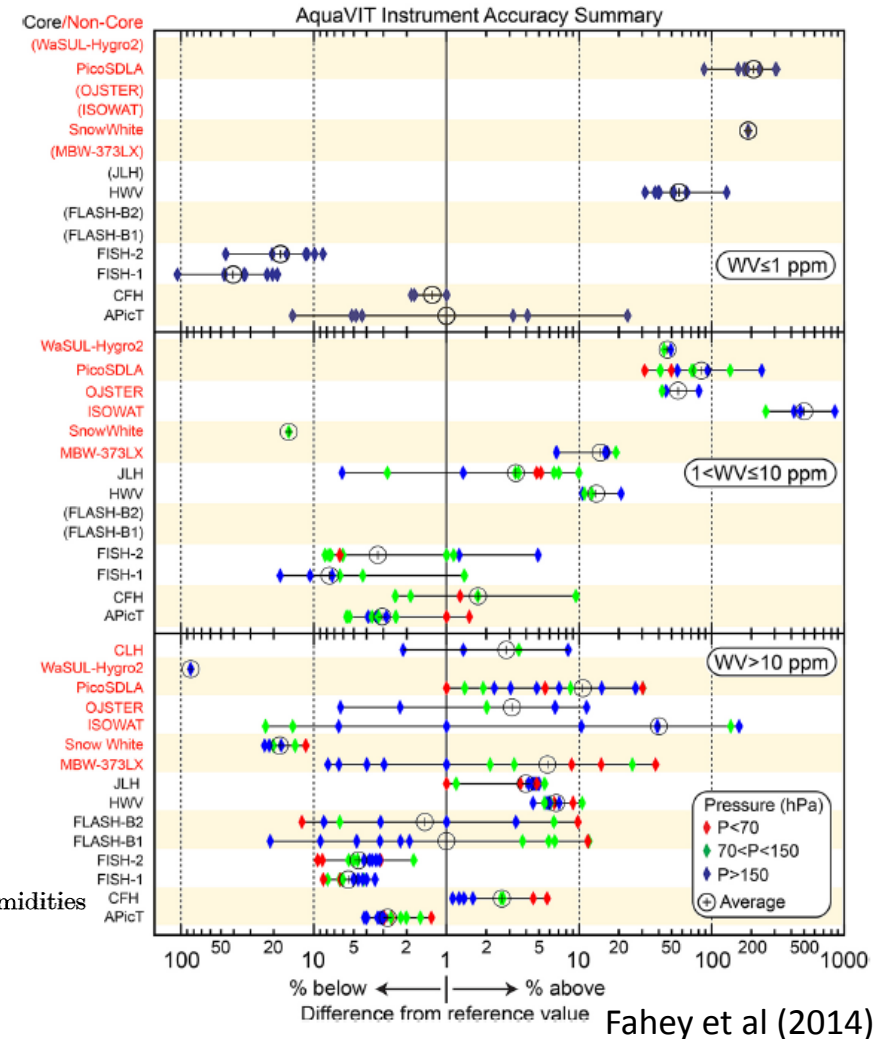


FIGURE 4. Frost-points and air temperatures observed on the first ascent when humidities were measured in the stratosphere, Boscombe Down, 22 December 1943.

M. B. Dobson, A. W. Brewer and B. M. Cwilong (1946)



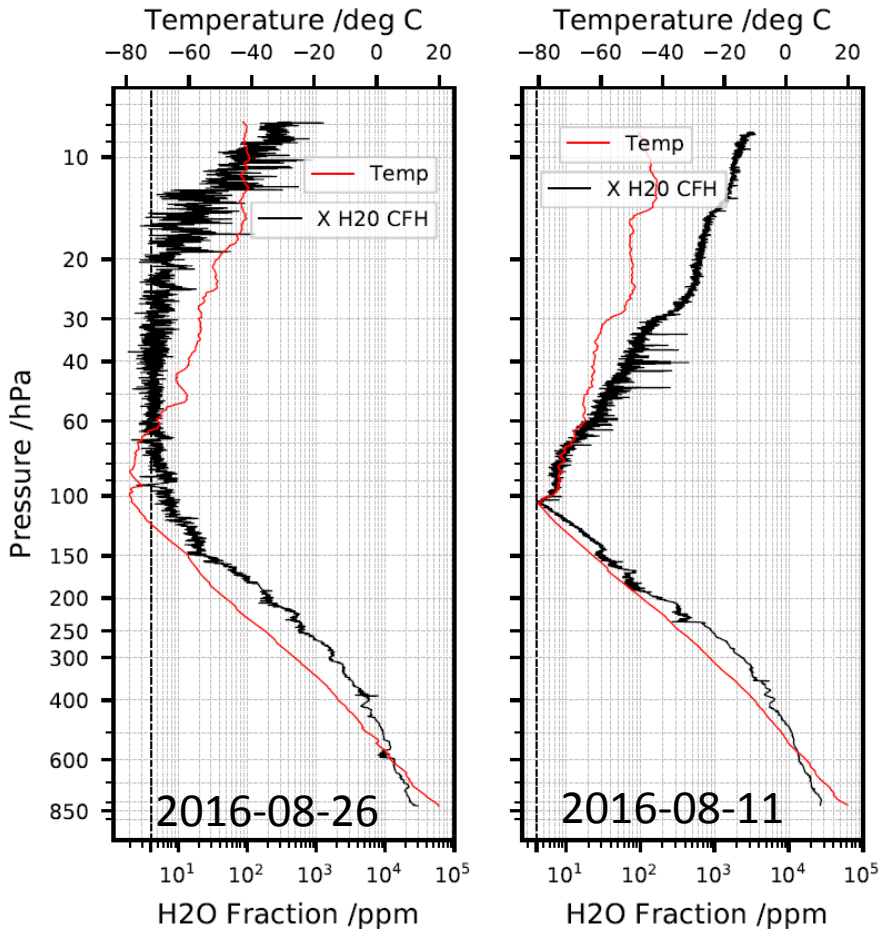
Uncertain future for Cryogenic Frost point Hygrometers



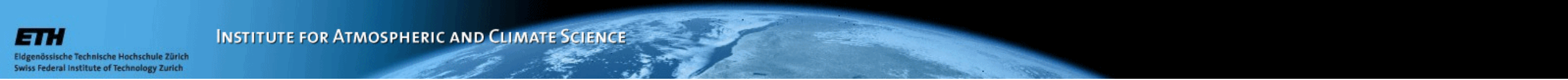
Kigali Amendment to Montreal protocol 2016



Complicated dry ice and R23 logistics in Dhulikel Nepal 2017



Temperature and water vapor mixing ratio from RS41 + CFH balloon flights in Nainital India 2016



Outline

- Instrument Concept
- PCFH: 1st prototype
- PCFH: Outlook
- System identification
 - Cold side
 - Hot side
 - Reference Surface
 - Reflectance
- Take home message

Instrument concept

- **Double Instrument**

- 2 identical optical assemblies
- 2 individual flow tubes
- Single polystyrene housing

- **Double stage Peltier cooling**

- Lowest reachable T_{cold}
at given T_{hot} :

T_{hot}	T_{cold}	ΔT
-75°C	-110°C	35 K
-38°C	-92 °C	54 K
27°C	-59 °C	86 K

- **Light detection scheme**

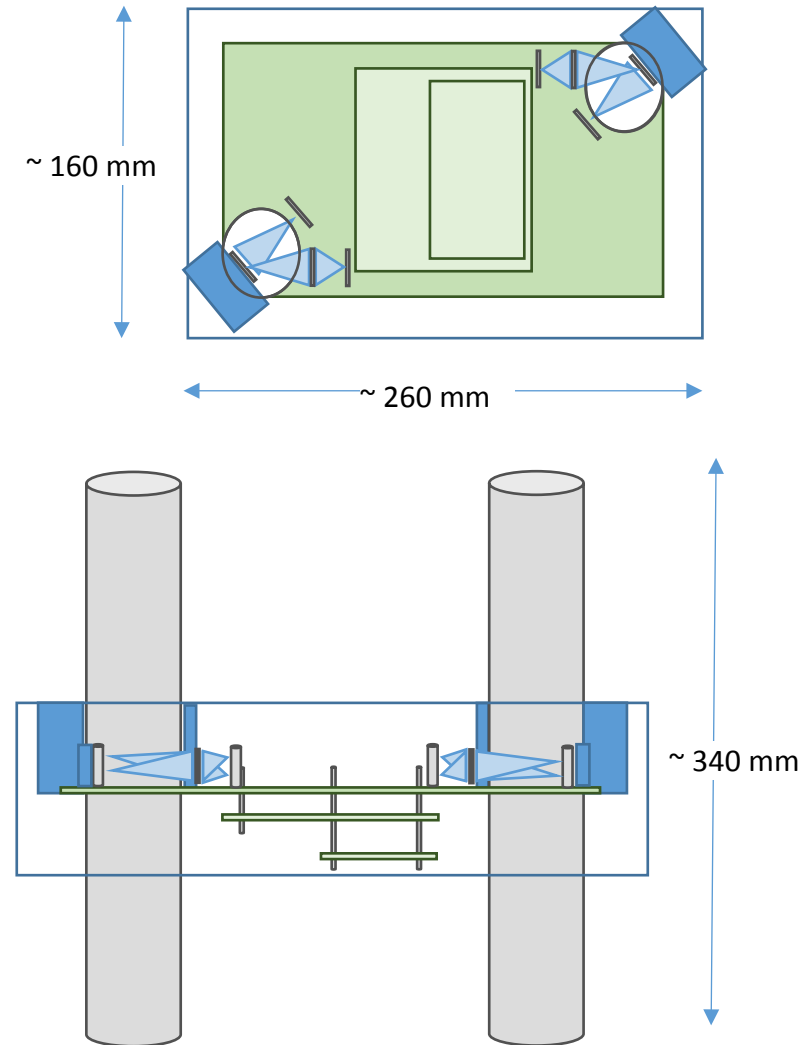
- Reference surface
- Kept 3°C to 5°C warmer than T_{air}

- **Thermocouples**

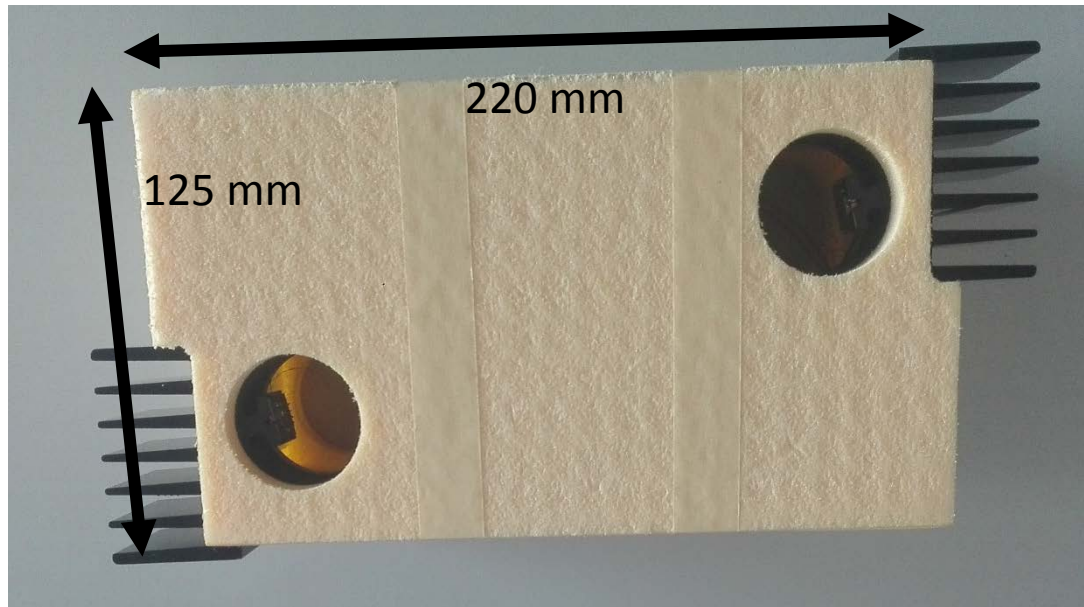
- Copper – constantan.
- Calibrated to Swiss national standards.
- Assembled precision 0.01 K and accuracy 0.1 K.
- Common reference temperature

- **Controller and Plant model**

- Linear Quadratic Regulator (LQR)
- Full state feedback offering superior robustness
- Replaces the common PID controller
- Mathematical/ physical description of the Instrument



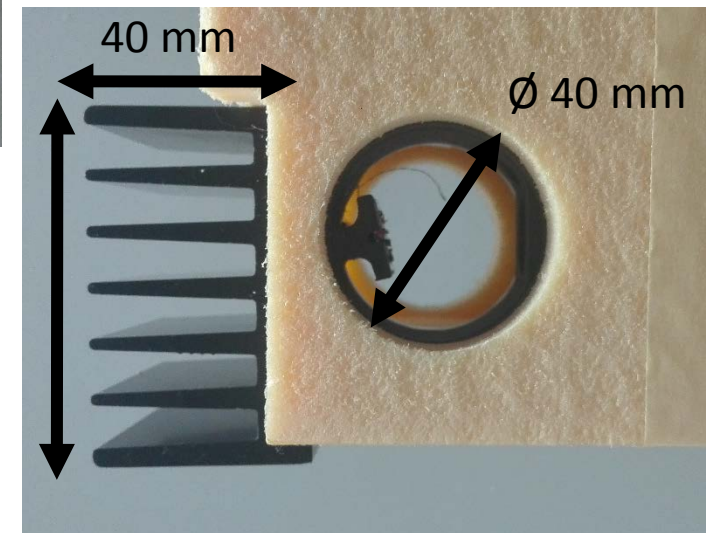
PCFH: 1st prototype



Box - top view

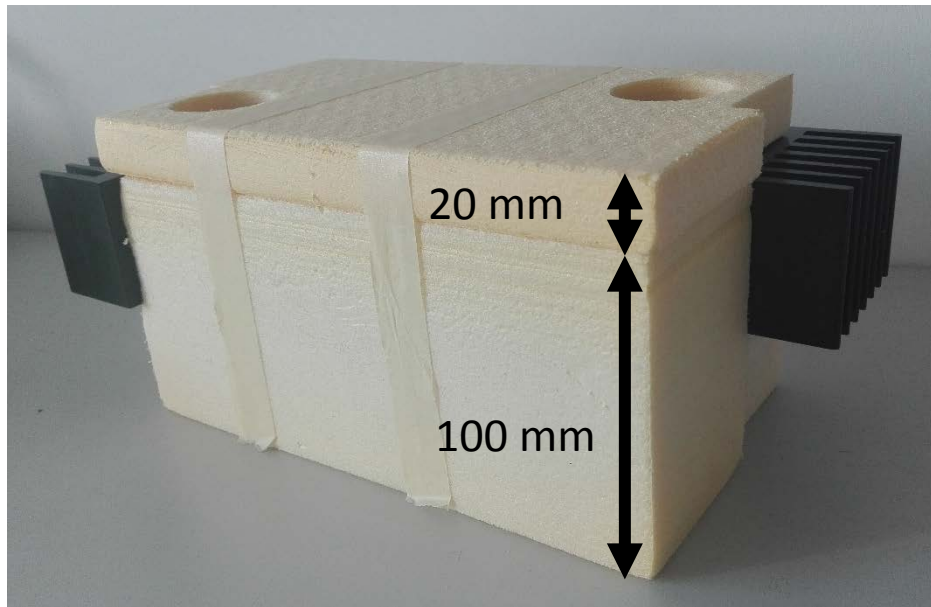
66 mm

Weight: ~800 g

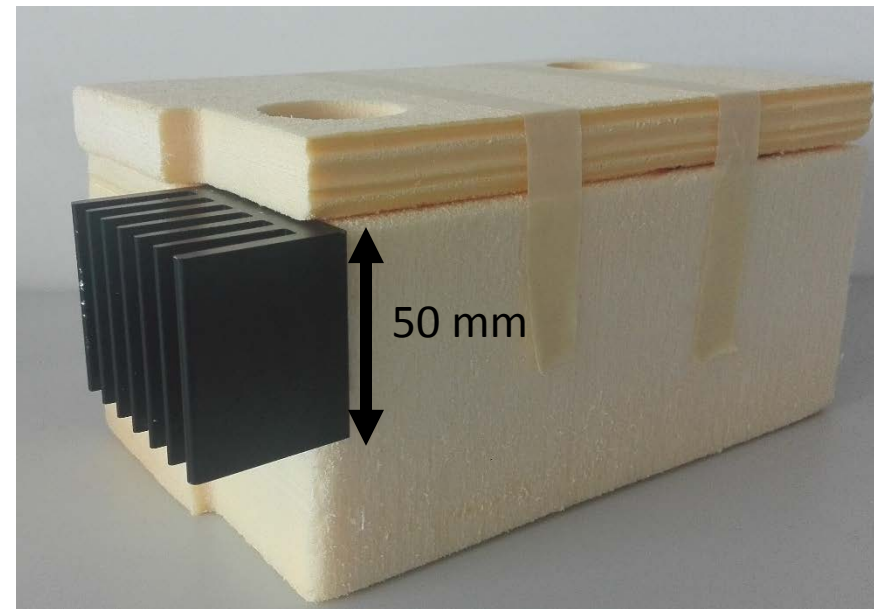


Inlet – detail view

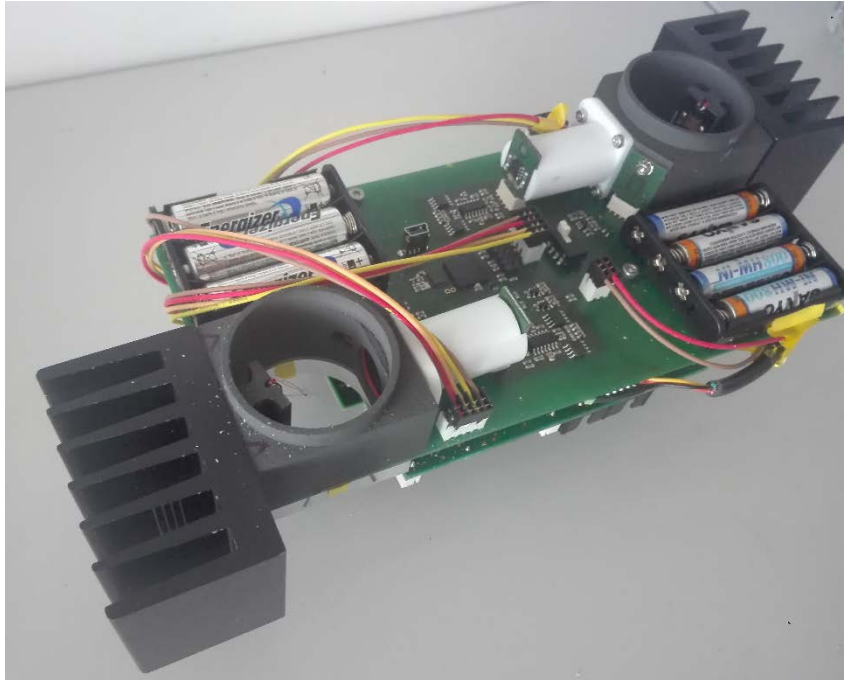
PCFH: 1st prototype



Box – side view



PCFH: 1st prototype



Mirror detail

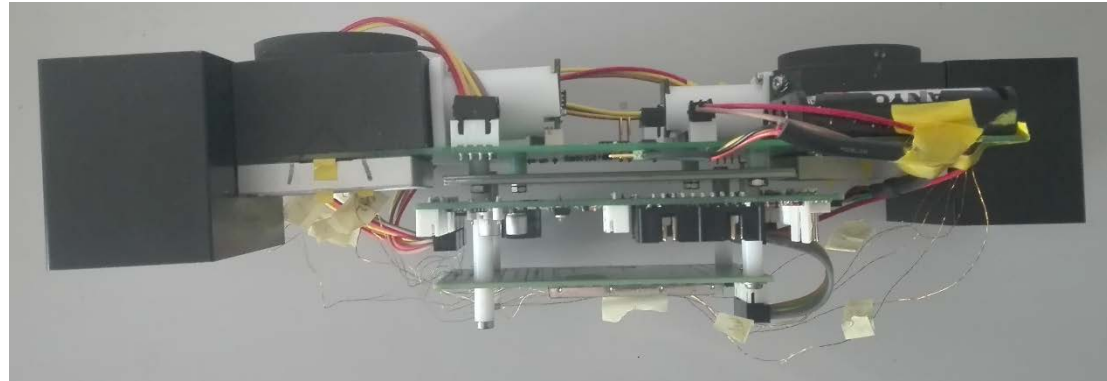


Sub-unit 2

Sub-unit 1

Top view

PCFH: 1st prototype



Side view

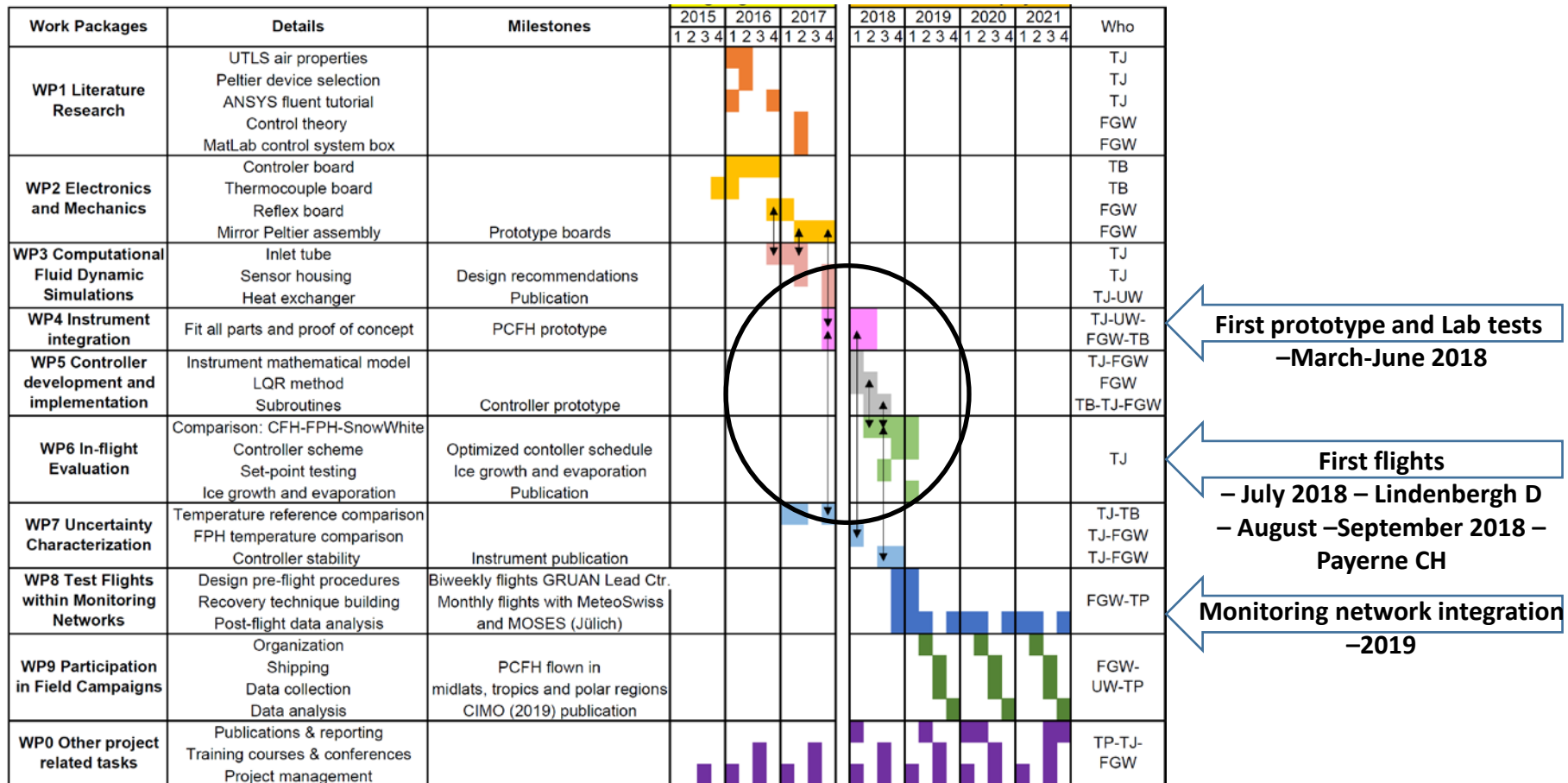


Bottom view



Bottom view - detail

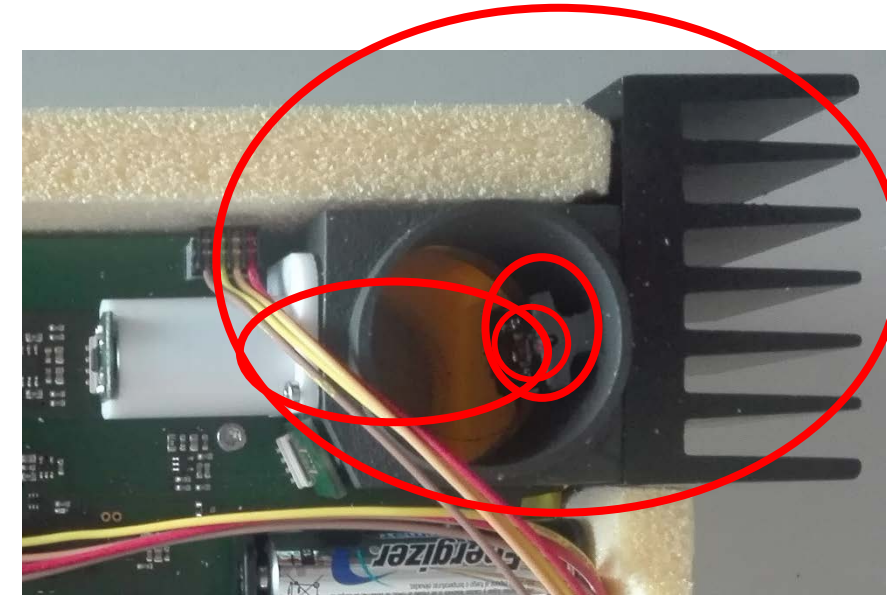
PCFH: Outlook



- GAW+ proposal accepted (for GRUAN integration of PCFH)
- Intern from June to October 2018

System Identification

- Black, Grey and White box models
- Combination of flows and reservoirs: dynamic behavior
- Thermodynamic system
- Differential equation form
- State variables, inputs and output
- Advantages: validate model at ground conditions
 - Extrapolate behavior in “atmospheric” conditions



System Identification – cold side

$$\dot{m}_{ice} \cdot C_{P_{ice}} \cdot T_C + ((m_{ice} \cdot C_{P_{ice}} + m_{air} C_{P_{air}}) \cdot \frac{dT_C}{dt} = -\dot{Q}_P + \frac{\dot{Q}_{JP}}{2} + \dot{Q}_{\Delta T} - \dot{Q}_C + \dot{Q}_L$$

$\dot{Q}_P = I_P \cdot \alpha_P \cdot T_C$ Heat removed by Peltier effect – α_P : Seebeck coefficient

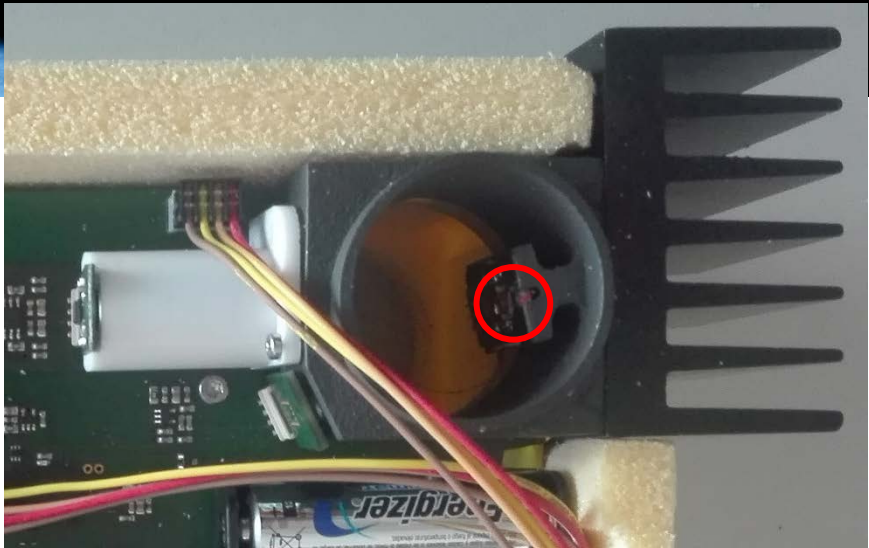
$\dot{Q}_{JP} = R_P \cdot I_P^2$ Joule heating by the Peltier

$\dot{Q}_{\Delta T} = K_P \cdot (T_H - T_C)$ Heat transfer from hot side to cold side due to temperature difference

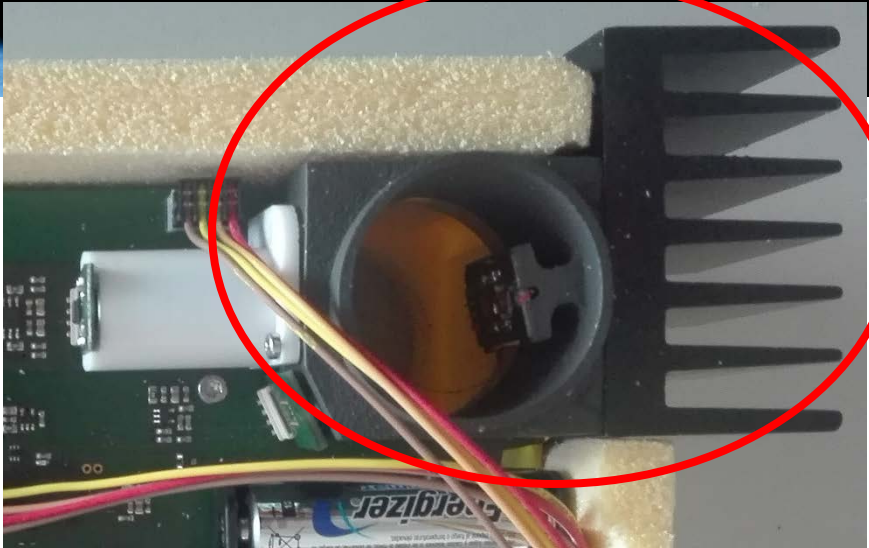
$\dot{Q}_C = (m_{air} \cdot C_{P_{air}}) \cdot (T_{air} - T_C)$ Heat removed or added from the ambient air

$\dot{Q}_L = \dot{m}_{ice} \cdot L_V$ Ice sublimation and condensation heat due to change in the mass of the ice layer

\dot{Q}_P (W)	\dot{Q}_{JP} (W)	$\dot{Q}_{\Delta T}$ (W)	\dot{Q}_C (W)	\dot{Q}_L (W)
0.65 – 2	0.7 – 1.7	0.16 – 0.33	$10^{-5} - 10^{-4}$	0.001 – 0.01



System Identification – hot side



$$(m_{MB} \cdot C_{P_{MB}} + m_{HS} \cdot C_{P_{HS}}) \cdot \frac{dT_H}{dt} = \frac{\dot{Q}_{JP}}{2} + \frac{\dot{Q}_{JRS}}{2} + \dot{Q}_{HS} + \dot{Q}_{inlet} + \dot{Q}_{inst} + \dot{Q}_{RHS}$$

$$\dot{Q}_{JP} = R_P \cdot I_P^2$$

Joule heating by the Peltier

$$\dot{Q}_{JRS} = R_{RS} \cdot I_H^2$$

Joule heating from the reference mirror board

$$\dot{Q}_{HS} = 7 \times \sqrt{k_{Al} \cdot A_c \cdot h_{HS} \cdot p} \cdot (T_H - T_{air}) \cdot \tanh(a \cdot L_c)$$

Forced convection from 7-fin heat sink too ambient air when base is at the temperature of the hot side.

$$\dot{Q}_{inlet} = h_{inlet} \cdot A \cdot (T_H - T_{air})$$

Forced convection from inside inlet tube

$$\dot{Q}_{inst} = h_{inst} \cdot A \cdot (T_H - T_{inst})$$

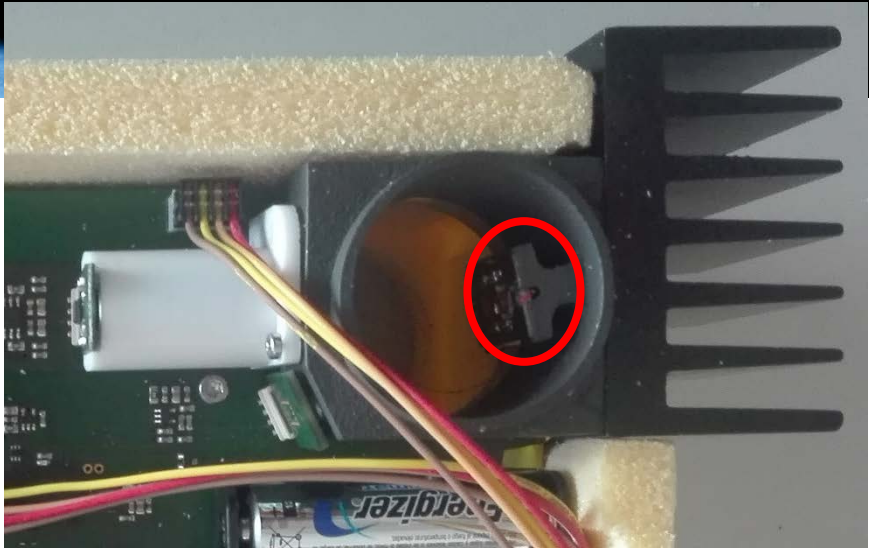
Natural convection from the interior of the instrument

$$\dot{Q}_{RHS} = \epsilon \cdot \sigma \cdot A_C \cdot (T_H^4 - T_{rad}^4)$$

Radiative heat transfer from the environment

\dot{Q}_{JP} (W)	\dot{Q}_{JRS} (W)	\dot{Q}_{HS} (W)	\dot{Q}_{inlet} (W)	\dot{Q}_{inst} (W)	\dot{Q}_{RHS} (W)
0.65 – 2	0 – 8	0 – 20	0 – 0.4	-0.5 – 0.2	1 – 6

System Identification – reference surface



$$m_{RS} \cdot c_{pRS} \cdot \frac{dT_{ReSur}}{dt} = \dot{Q}_{RS} + \frac{\dot{Q}_{JRS}}{2}$$

$\dot{Q}_{JRS} = R_{RS} \cdot I_H^2$

Heat added by joule effect from the reference mirror board

$\dot{Q}_{RS} = h_{inlet} \cdot A_{RS} \cdot (T_{RefSurf} - T_{air})$

Heat removed by convection by the air in the inlet tube



\dot{Q}_{JRS} (W)	\dot{Q}_{RS} (W)
0 – 8	0 – 0.2

System Identification – reflectance



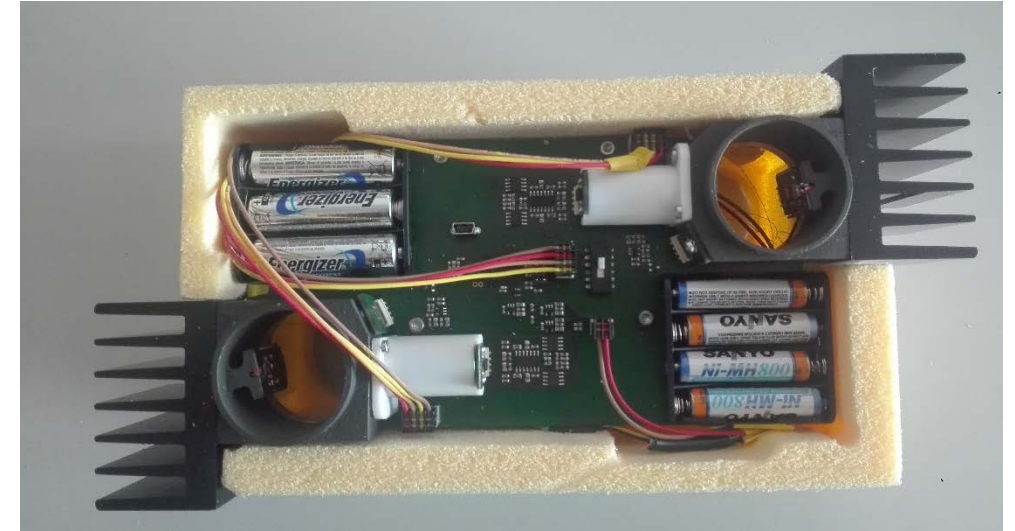
$$\frac{dR}{dt} = -A \cdot \dot{m} = -B(T_{fr}) \cdot (T_{fr} - T_c)$$

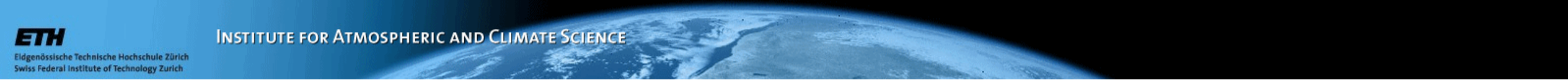
$$T_{fr} < T_c : \dot{m} < 0 : \frac{dR}{dt} > 0$$

$$T_c < T_{fr} : \dot{m} > 0 : \frac{dR}{dt} < 0$$

Take home message

- New Peltier Cooled Frost point Hygrometer - PCFH
- Developed as a GRUAN worthy instrument
 - Temperature measurement traceable back to measurement standards
 - Transparency of implementation
- Flight tests to start in July 2018
- Novelty on the controller development
 - System Identification and Modelling





Thank you for your attention

Questions?